

# Low exergy systems for heating and cooling



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# IEA ECBCS Annex 49

## Low Exergy Systems for High-Performance Buildings and Communities

- Exergy is a measure of energy quality

<i>Exergy/energy ratio (<math>T_0=20^\circ\text{C}</math>)</i>	
<i>Mechanical energy</i>	<i>1.00</i>
<i>Electrical energy</i>	<i>1.00</i>
<i>Nuclear fuel</i>	<i>1.00</i>
<i>Solar radiation</i>	<i>0.95</i>
<i>Fossil fuels</i>	<i>0.90</i>
<i>Heat at <math>100^\circ\text{C}</math></i>	<i>0.21</i>
<i>Heat at <math>40^\circ\text{C}</math></i>	<i>0.06</i>

- Although building heating and cooling require low valued energy, climatisation is usually provided with high exergy sources!

# IEA ECBCS Annex 49

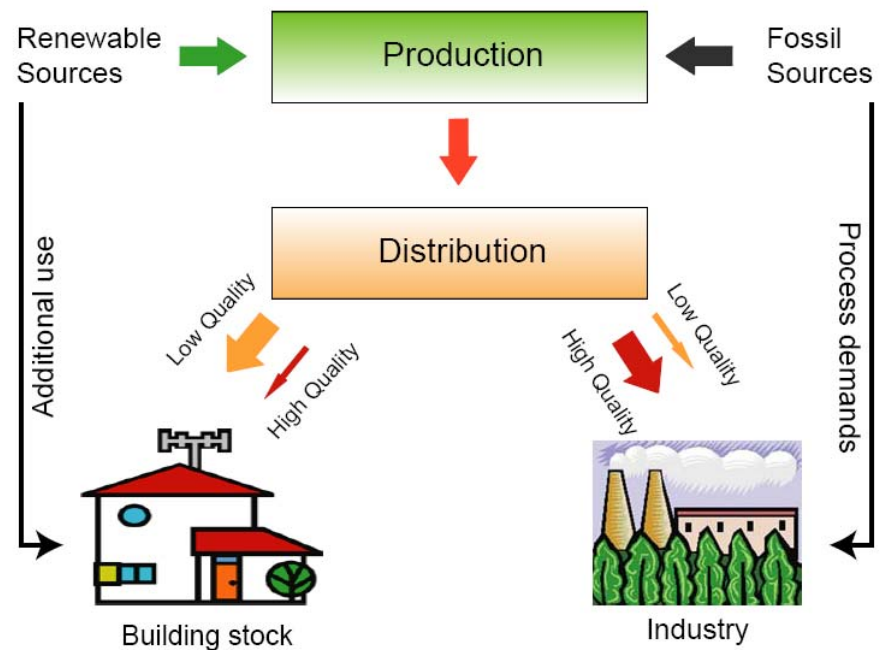
## Low Exergy Systems for High-Performance Buildings and Communities

### The LowEx approach:

- matching the quality levels of the energy supply and demand for the building stock
- minimising exergy consumption in the building systems

### The Annex 49 project:

- developing exergy analysis methodologies and tools
- Pinpointing and promoting exergy efficient community supply systems and building technologies



# Energy\exergy analysis of heating/cooling options in Italy

Comparison among different commercial climatisation options

## Methodology:

- Steady state calculation of exergy efficiency with design temperatures seasonally and locally varying:
  - Heating  $T_U = 20^\circ\text{C}$   $T_o = -5^\circ\text{C}$  (Mi),  $5^\circ\text{C}$  (Pa)
  - Cooling  $T_U = 26^\circ\text{C}$   $T_o = 32^\circ\text{C}$  (Mi, Pa)
- Energy generation (plant) and heating/cooling supply (system) levels
- System size referring to a single residential unit

# Energy\exergy analysis of heating/cooling options in Italy

## Plant level:

- Electricity/heat generation: Italian electricity mix, CCGT, large and mini CHP, PV
- Performance:
  - Energy efficiency  $\eta_1$
  - Exergy efficiency  $\varepsilon_1 = \frac{Ex_U}{Ex_C}$

## System level:

- Performance:
  - Energy efficiency  $\eta_2$  (COP)
  - Exergy efficiency  $\varepsilon_2 = Ex_U / Ex_C =$   
 $= Q_U \left| 1 - \frac{T_0}{T_U} \right| / Ex_C$

## Entire energy chain performance:

- Overall energy efficiency  
 $\eta = \eta_1 \eta_2$
- Overall exergy efficiency  
 $\varepsilon = \varepsilon_1 \varepsilon_2$

# Energy\exergy analysis of heating/cooling options in Italy

System level: example of analysis

- comparison among heat pumps ( $T_0 = -5^\circ\text{C}$ )

system	<b>COP</b>	$\varepsilon$	<b>COP/ COP<sub>0</sub></b>	$\varepsilon/\varepsilon_0$
<b>Air-to-air</b> heat pump	<b>2.82</b>	<b>0.24</b>	<b>100 %</b>	<b>100 %</b>
<b>Air-to-water</b> heat pump	<b>1.79</b>	<b>0.15</b>	<b>64 %</b>	<b>64 %</b>
<b>Ground source</b> heat pump	<b>4.50</b>	<b>0.32</b>	<b>160 %</b>	<b>132 %</b>

$$\frac{\varepsilon_{GSHP}}{\varepsilon_{ASHP}} = \frac{COP_{GSHP}}{COP_{ASHP}} \frac{1}{1 + (COP_{GSHP} - 1) \left(1 - \frac{T_0}{T_G}\right)}$$

“thermal pollution” entropy production

- comparison between absorption chillers ( $T_0 = 32^\circ\text{C}$ )

system	<b>COP</b>	$\varepsilon$	<b>COP/COP<sub>0</sub></b>	$\varepsilon/\varepsilon_0$
<b>Single</b> stage	<b>0.7</b>	<b>0.103</b>	<b>100 %</b>	<b>100 %</b>
<b>Double</b> stage	<b>1.1</b>	<b>0.079</b>	<b>157 %</b>	<b>77 %</b>

different temperature levels in input!

# Energy\exergy analysis of heating/cooling options in Italy

## Entire energy chain analysis: heating

## Heating options (Milano)

<i>source</i>	<i>generation</i>	$\eta_1$	$\varepsilon_1$	<i>system</i>	$\eta_2$	$\varepsilon_2$	$\eta$	$\varepsilon$
RE equivalent	waste heat recovery	1	1	district heating	0.9	0.319	<b>0.90</b>	<b>0.319</b>
RE equivalent	small size cogeneration	3.30	0.79	district heating	0.9	0.319	<b>2.97</b>	<b>0.253</b>
fossil+RE	combined cycle	0.55	0.55	ground source heat pump	4.5	0.318	<b>2.48</b>	<b>0.175</b>
fossil	combined cycle	0.55	0.55	air-to-air heat pump	2.82	0.240	<b>1.55</b>	<b>0.132</b>
fossil+RE	Italian electrical mix	0.36	0.36	ground source heat pump	4.5	0.318	<b>1.62</b>	<b>0.114</b>
RE equivalent	large size cogeneration	1.31	0.32	district heating	0.9	0.319	<b>1.18</b>	<b>0.100</b>
fossil	Italian electrical mix	0.36	0.36	air-to-air heat pump	2.82	0.240	<b>1.01</b>	<b>0.086</b>
fossil	√			condensing gas boiler + radiant panels	1.05	0.085	<b>1.05</b>	<b>0.085</b>
fossil	combined cycle	0.55	0.55	air-to-water heat pump	1.79	0.153	<b>0.98</b>	<b>0.084</b>
fossil	√			condensing gas boiler + radiator	0.98	0.075	<b>0.98</b>	<b>0.075</b>
fossil	√			gas boiler + radiator	0.86	0.067	<b>0.86</b>	<b>0.067</b>
fossil	Italian electrical mix	0.36	0.36	air-to-water heat pump	1.79	0.153	<b>0.64</b>	<b>0.055</b>
RE	photovoltaic	0.15	0.16	ground source heat pump	4.5	0.318	<b>0.68</b>	<b>0.050</b>
RE	photovoltaic	0.15	0.16	air-to-air heat pump	2.82	0.240	<b>0.42</b>	<b>0.038</b>
fossil+RE	√			solar collectors+condens. gas boiler+radiant panels	0.38	0.033	<b>0.38</b>	<b>0.033</b>
RE	photovoltaic	0.15	0.16	air-to-water heat pump	1.79	0.153	<b>0.27</b>	<b>0.024</b>

▪ moving from Milano to Palermo affects slightly the order of the list

▪ most exergy efficient solutions: using waste or cogeneration heat, energy efficient heat pumps +CCGT

▪ fully renewable solutions (solar systems) are in general the less exergy efficient....but renewable!

# Energy\exergy analysis of heating/cooling options in Italy

## Entire energy chain analysis: cooling

<i>source</i>	<i>generation</i>	$\eta_1$	$\varepsilon_1$	<i>system</i>	$\eta_2$	$\varepsilon_2$	$\eta$	$\varepsilon$
RE equivalent	waste heat recovery	1	1	single stage absorption chiller	0.7	0.103	<b>0.7</b>	<b>0.103</b>
RE equivalent	waste heat recovery	1	1	double stage absorption chiller	1.1	0.079	<b>1.1</b>	<b>0.079</b>
fossil+RE	combined cycle	0.55	0.55	direct ground cooling	10	0.123	<b>5.50</b>	<b>0.068</b>
RE equivalent	small size cogeneration	3.30	0.45	single stage absorption chiller	0.7	0.103	<b>2.31</b>	<b>0.046</b>
fossil+RE	Italian electrical mix	0.36	0.36	direct ground cooling	10	0.123	<b>3.60</b>	<b>0.044</b>
fossil+RE	combined cycle	0.55	0.55	ground source heat pump	4.5	0.072	<b>2.48</b>	<b>0.040</b>
fossil	combined cycle	0.55	0.55	air-to-air heat pump	3.46	0.069	<b>1.90</b>	<b>0.038</b>
fossil	combined cycle	0.55	0.55	air-to-water heat pump	3.25	0.065	<b>1.79</b>	<b>0.036</b>
fossil+RE	Italian electrical mix	0.36	0.36	ground source heat pump	4.5	0.072	<b>1.62</b>	<b>0.026</b>
fossil	Italian electrical mix	0.36	0.36	air-to-air heat pump	3.46	0.069	<b>1.25</b>	<b>0.025</b>
fossil	Italian electrical mix	0.36	0.36	air-to-water heat pump	3.25	0.065	<b>1.17</b>	<b>0.023</b>
RE	photovoltaic	0.15	0.16	direct ground cooling	10	0.123	<b>1.50</b>	<b>0.019</b>
fossil	large size cogeneration	1.31	0.18	single stage absorption chiller	0.7	0.103	<b>0.92</b>	<b>0.018</b>
RE	photovoltaic	0.15	0.16	ground source heat pump	4.5	0.072	<b>0.68</b>	<b>0.011</b>
RE	photovoltaic	0.15	0.16	air-to-air heat pump	3.46	0.069	<b>0.52</b>	<b>0.011</b>
RE	photovoltaic	0.15	0.16	air-to-water heat pump	3.25	0.065	<b>0.49</b>	<b>0.010</b>
RE	Solar collector	0.40	0.06	single stage absorption chiller	0.7	0.103	<b>0.28</b>	<b>0.006</b>

■ **cooling exergy efficiency generally lower than heating!**

■ **most exergy efficient solutions: waste heat + absorption chillers and CCGT+ direct ground cooling**

# Dynamic vs steady state analysis

Preliminary comparison between steady state and dynamic evaluation of exergy efficiency for some heating/cooling systems

Dynamic simulation of the heating and cooling demand of a residential unit

	$\varepsilon$ steady	$\varepsilon$ dynamic
Air source reversible heat pump	$COP(T_0) \cdot \left(1 - \frac{T_0}{T_U}\right)$	$\left\langle COP(T_0(t)) \cdot \left(1 - \frac{T_0(t)}{T_U}\right) \right\rangle$
Condensing boiler	$\eta \cdot \left(1 - \frac{T_0}{T_U}\right)$	$\eta \left\langle \left(1 - \frac{T_0(t)}{T_U}\right) \right\rangle$
Direct ground cooling system	$COP \cdot \frac{\left 1 - \frac{T_0}{T_U}\right }{\left 1 - \frac{T_0}{T_G}\right }$	$COP \cdot \left\langle \frac{\left 1 - \frac{T_0(t)}{T_U}\right }{\left 1 - \frac{T_0(t)}{T_G}\right } \right\rangle$

# Dynamic vs steady state analysis

<i>Milano</i>	$\varepsilon$ steady $T_0=T_{design}$	$\varepsilon$ steady $T_0=\langle T_0(t) \rangle$	$\varepsilon$ dynamic
Air source reversible heat pump (heating)	0.184	0.157 (-15 %)	0.157 (-15 %)
Condensing boiler	0.079	0.061 (-23 %)	0.062 (-22 %)
Air source reversible heat pump (cooling)	0.052	√	0.036 (-31 %)
Direct ground cooling system	0.201	√	0.117 (-42 %)

- Significant difference between steady and dynamic values, but if  $T_0$  is a monthly mean instead of a design value ....

- The difference depends on the system

- Further investigations are necessary...